

# Staffa Dual Displacement Hydraulic Motor



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#### 1. GENERAL DESCRIPTION

Kawasaki "Staffa" high torque, low speed radial piston motors use hydrostatic balancing techniques to achieve high efficiency, combined with good breakout torque and smooth running capability.

The HMC series dual displacement models have two pre-set displacements which can be chosen from a wide range to suit specific application requirements. The displacements are hydraulically selected by a directional control valve which can be remote from, or mounted directly on, the motor. Displacements can be changed when the motor is running.

The range of HMC motors extends from the HMC010 of 202 cm³ (12.3 in³) to the HMC325 of 5330 cm³ (325 in³) displacement.

These motors are also available in a continuously variable version using either hydro-mechanical or electro-hydraulic control methods.

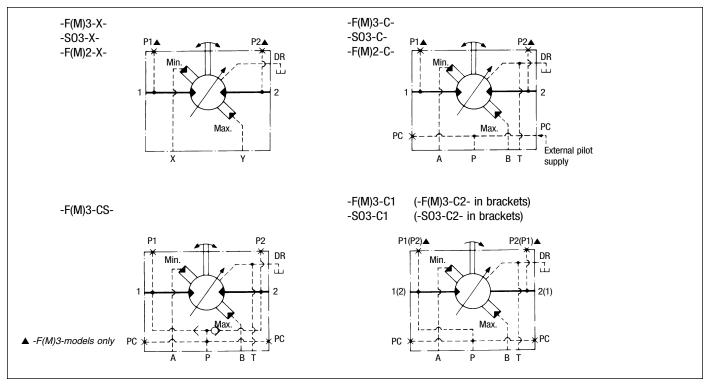
Other mounting options are available on request to match many of the competitor interfaces.

The HMC030 is one of 8 frame sizes and is capable of developing torques up to 1720 Nm (1270 lbf ft) with a continuous output power of 60 kW (80 hp).

The Kawasaki "Staffa" range also includes fixed displacement motors, plus matching brakes and gearboxes to extend the torque range.

#### 2. FUNCTIONAL SYMBOLS

All model types with variants in model code positions **6** & **7**.



#### 3. MODEL CODE

Features shown in brackets () may be left blank according to requirements. All other features must be specified.

(F\*\*)-HM(\*)C030-\*\*-\*\*-\*\*-(T\*)-30-(PL\*\*)

1 2 3 4 5 6 7 8 9 10

## 1 FLUID TYPE

Blank = Petroleum oil

F3 = Phosphate ester (HFD fluid) F11 = Water-based fluids (HFA, HFB and HFC)

## 2 MODEL TYPE

Blank = Standard ("HMC")

M = To NCB (UK) specification 463/1981 ("HMMC")

## 3 SHAFT TYPE

P\* = Cylindrical shaft with parallel key

S\* = Cylindrical, 17 splines to BS 3550

 $Z^*$  = Cylindrical shaft to DIN 5480 (W55 x 3 x 7h)

\* For installations where shaft is vertically upwards specify "V" after shaft type letter to ensure that additional high level drain port is provided.

## 4 HIGH DISPLACEMENT CODE

15 to 30 in<sup>3</sup>. in 3 in<sup>3</sup> steps

## 5 LOW DISPLACEMENT CODE

21 to 03 in<sup>3</sup>, in 3 in<sup>3</sup> steps

## **6** MAIN PORT CONNECTIONS

Models with 2<sup>1</sup>/<sub>4</sub>" distributor valve◆ F2 = SAE 1" 4-bolt (UNC) flange

FM2 = SAE 1" 4-bolt (metric) flange

Models with 3" distributor valve

SO3 = 6-bolt (UNF) flange (Staffa original valve housing)

F3 = SAE 1<sup>1</sup>/<sub>4</sub>" 4-bolt (UNC) flange

FM3 = SAE 1<sup>1</sup>/<sub>4</sub>" 4-bolt (metric) flange

◆ Gives minimum overall length of HMC030 motor

## 7 DISPLACEMENT CONTROL PORTS (AND SHUTTLE VALVE)

Threaded ports/bi-directional shaft rotation:

 $X = X \text{ and } Y \text{ ports } G^{1/4}$ " (BSPF to ISO 228/1)

ISO 4401 size 03 mounting face/bidirectional shaft rotation:

C = No shuttle valve

CS■ = With shuttle valve

ISO 4401 size 03 mounting face/uni-directional shaft rotation (viewed on shaft end):

C1▲ = Control pressure from main port 1 (shaft rotation clockwise with flow into port 1)

C2▲ = Control pressure from main port 2 (shaft rotation counter-clockwise with flow into port 2)

■ Not available with "S03" and "F(M)2" type main port connections 6

▲ Not available with "F(M)2" type main port connections 6

## 8 TACHO/ENCODER DRIVE

T = Staffa original tacho drive

T1 = Suitable for Hohner 3000 series encoders. (Encoder to be ordered separately)

Omit if not required.

## 9 DESIGN NUMBER, 30 SERIES

Subject to change. Installation and performance details remain unaltered for design numbers 30 to 39 inclusive.

## 10 SPECIAL FEATURES

PL\*\* = non-catalogued features,

e.q.:

High pressure shaft seals
Alternative port connections
Stainless steel shaft sleeves
Alternative encoder and tacho drives
Motor valve housing orientation

Shaft variants

Special paint

\*\* Number assigned as required to specific customer build.

#### 4. PERFORMANCE DATA

Performance data is valid for Staffa HMC030 motors fully run in and operating with petroleum oil. Leakage values are at fluid viscosity of 50 cSt (232 SUS).

#### **MOTOR SELECTION**

Use table 1 to select appropriate displacements for each application.

Refer to table 2 for pressure and speed limits when using fire-resistant fluids.

**TABLE 1** 

Displacement code* (Model code positions											
4 & 5)		30	27	24	21	18	15	12	09	06	03
Displacement	cm <sup>3</sup>	492	442	393	344	295	246	197	147	98	49
volume/r	in <sup>3</sup>	30	27	24	21	18	15	12	9	6	3
Average actual	Nm/bar	6,86	6,08	5,30	4,59	3,88	3,20	2,51	1,83	1,15	0,44
running torque	lbf ft/psi	0.349	0.309	0.269	0.233	0.197	0.163	0.128	0.093	0.058	0.022
Max. continuous											
speed	r/min	450	500	525	550	575	600	600	600	600	1000
Max. continuous	kW	60	60	55	49	42	35	27	20	10	0
output	hp	80	80	74	66	56	47	37	27	13	0
Max. intermittent	kW	66	66	61	55	48	41	32	24	13	0
output	hp	88	88	82	74	64	55	43	32	17	0
Max. continuous	bar	207	207	207	207	207	207	207	207	207	17◆
pressure	psi	3000	3000	3000	3000	3000	3000	3000	3000	3000	250◆
Max. intermittent	bar	241	241	241	241	241	241	241	241	241	17◆
pressure	psi	3500	3500	3500	3500	3500	3500	3500	3500	3500	250◆

<sup>\*</sup> Intermediate displacements are made available to special order.

TABLE 2

Fluid type	Pressure, ba Continuous	r (psi) Intermittent	Max. speed r/min
HFA, 5/95% oil-in-water emulsion	103 (1500)	138 (2000)	50% of limits for petroleum oil
HFB, 60/40% water-in-oil emulsion	138 (2000)	172 (2500)	As for petroleum oil
HFC, water glycol	103 (1500)	138 (2000)	50% of limits for petroleum oil
HFD, phosphate ester	207 (3000)	241 (3500)	As for petroleum oil

#### **RATING DEFINITIONS**

#### CONTINUOUS RATING

For continuous duty the motor must be operating within each of the maximum values for speed, pressure and power as specified for each displacement code.

#### • INTERMITTENT RATING

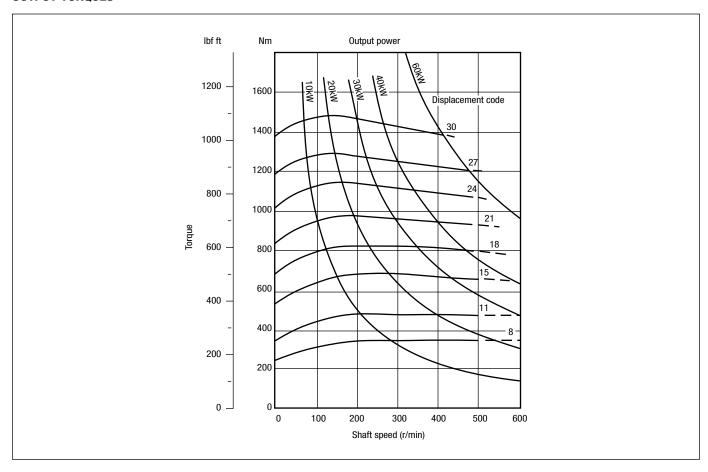
Operation within the intermittent power rating (up to the maximum continuous speed) is permitted on a 15% duty basis, for periods up to 5 minutes maximum.

## ● INTERMITTENT MAX. PRESSURE Up to 241 bar (3500 psi) is allowable on the following basis:

- (a) Up to 100 r/min: 15% duty for periods up to 5 minutes maximum.
- (b) Over 100 r/min: 2% duty for periods up to 30 seconds maximum.

<sup>◆</sup> See "Small displacements" page 7 for information about higher pressure applications.

#### **OUTPUT TORQUES**



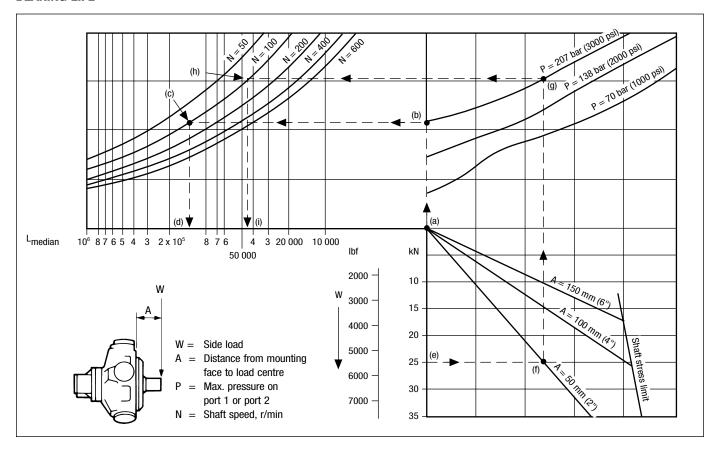
The torque curves indicate, for each displacement, the maximum output torque of the motor with an inlet pressure of 207 bar (3000 psi) and zero output pressure. High return line pressures will reduce the torque for any given pressure differential.

The solid line portion of each curve indicates the levels of maximum torque and speed that are permitted on a "continuous" basis.

The dotted portion of each curve indicates the levels of torque and speed at which the motor can operate at an "intermittent" rating.

The starting torques shown on the graph are average and will vary with crankshaft angle.

#### **BEARING LIFE**



The nomograph allows the median ▲ bearing life to be determined for conditions of:

- 1. No side load and no axial thrust
- 2. Side load and no axial thrust
- ▲ To determine L10 life predictions per ISO 281-1-1977 multiply the median figure by 0.2.

For more precise life prediction, or where axial trusts are incurred, a computer analysis can be provided by Kawasaki on receipt of machine duty cycle.

	HMC030
Example 1 (follow chain dotted line):	
Side load (W)	a) 0
System pressure (P)	b) 207 bar (3000 psi)
Speed (N)	c) 100 r/min
Median bearing life	d) 136 000 hrs
L10 bearing rating = median $x = 0.2$	27 200 hrs
Example 2 (follow chain dotted line):	
Side load (W)	e) 25 kN (5620 lbf)
Load offset (A) from motor mounting face	f) 50 mm (2.0 in)
System pressure (P)	g) 207 bar (3000 psi)
Speed (N)	h) 100 r/min
Median bearing life	i) 45 500 hrs
L10 bearing rating = median x 0.2	9100 hrs

#### • SHAFT STRESS LIMIT

The shaft stress limit in the nomograph is based on the fatigue rating of the shaft. Infrequent loading above these limits may be permitted; consult Kawasaki.

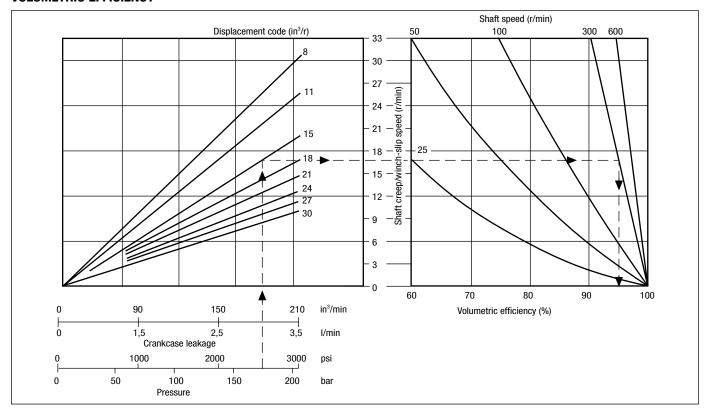
#### **VOLUMETRIC EFFICIENCY**

The nomograph on page 7 enables the average volumetric efficiency, crankcase (drain) leakage and "winch slip"/shaft creep speed to be estimated.

Example (follow chain dotted line): Given:

1.	Pressure17	5 bar (2500 psi)
2.	Displacement code	15 (in <sup>3</sup> /r)
3.	Speed	300 r/min
То	obtain:	
4.	Volumetric efficiency	94.8%
5.	Crankcase leakage .	3,0 l/min
		(180 in <sup>3</sup> /min)
6	Shaft creen sneed	17 r/min

The shaft creep occurs when the load attempts to rotate the motor against closed ports as may occur, for example, in winch applications.



# 5. CIRCUIT AND APPLICATION NOTES

#### **DISPLACEMENT SELECTION**

To select either displacement, a pressure at least equal to 2/3 of the motor inlet/outlet pressure (whichever is higher) is required. In most applications the motor inlet pressure will be used.

For inlet/outlet pressures below 3,5 bar (50 psi) a minimum control pressure of 3,5 bar (50 psi) is required. In the event of loss of control pressure the motor will shift to its highest displacement.

For rapid reversing applications it is recommended to externally source the control oil supply direct from the system pump (use displacement control type "X" or "C" - not "CS", "C1" or "C2" - in model code position \(\bar{\mathbb{Z}}\)).

#### STARTING TORQUES

The starting torques shown on the graph on page 5 are average and will vary with system parameters. For motors with low displacement below 12 in<sup>3</sup> and starting under load it is recommended to select high displacement for start-up.

#### **LOW SPEED OPERATION**

(High displacement mode)
Minimum operating speeds are
determined by load conditions
(load inertia, drive elasticity, etc.)
For operation at speeds below
3 r/min consult Kawasaki.

#### **SMALL DISPLACEMENTS**

(3 in<sup>3</sup> and below)

The pressures given in the table on page 4 for displacement code "03" (and below) are based on 1000 r/min output shaft speed. These pressures can be increased for shaft speeds less than 1000 r/min; consult Kawasaki for details.

In addition to 3 in<sup>3</sup>, a zero swept volume displacement (for free wheeling requirements) is available on request, subject to Kawasaki approving the application.

#### **HIGH BACK PRESSURE**

When both inlet and outlet ports are pressurized continuously, the lower pressure in one port must not exceed 70 bar (1000 psi). Consult Kawasaki on applications beyond this limit. Note that high back pressures reduce the effective torque output of the motor.

## **BOOST PRESSURE**

When operating as a motor the outlet pressure should equal or exceed the crankcase pressure. If pumping occurs (i.e. overrunning loads) then a positive pressure, "P", is required at the motor ports. Calculate "P" (bar/psi) from the appropriate formula:

$$P (bar) + 1 + \frac{N^2 \times V^2}{D_{bar}} + C$$

#### Where:

C = crankcase pressure, bar

D = see table

N = speed, r/min

 $V = displacement, cm^3/r$ 

P (bar) = 
$$14.5 + \frac{N^2 \times V^2}{D_{DSi}} + C$$

#### Where:

C = crankcase pressure, psi

D = see table

N = speed. r/min

 $V = displacement, cm^3/r$ 

Port connection type	D value
F3 FM3, S03	$D_{\text{bar}} = 7.5 \times 10^9$ $D_{\text{bar}} = 2.0 \times 10^6$
F2, FM2	$D_{bar} = 3.7 \times 10^9$ $D_{bar} = 9.5 \times 10^5$

The flow rate of oil needed for the make-up system can be estimated from the crankcase leakage figure (see Volumetric Efficiency graph above) plus an allowance for changing displacement; e.g. to change high to low in 0,2 sec requires 11 l/min (2.0 USgpm).

Allowance should be made for other system losses and also for "fair wear and tear" during the life of the motor, pump and other system components.

#### **COOLING FLOW**

Operation within the continuous ratings does not require any additional cooling.

For operating conditions above "continuous", up to the "intermittent" ratings, additional cooling oil may be required. This can be introduced through the spare crankcase drain holes, or in special cases through the valve spool end cap. Consult Kawasaki about such applications.

#### **MOTOR CASING PRESSURE**

With the standard shaft seal fitted, the motor casing pressure should not exceed 3,5 bar (50 psi). On installations with long drain lines a relief valve is recommended to prevent overpressurizing the seal.

#### Notes:

- The casing pressure at all times must not exceed either the motor inlet or outlet pressure.
- High pressure shaft seals are available to special order for casing pressures of: Continuous: 10 bar (150 psi) Intermittent: 15 bar (225 psi)
- 3. Check installation dimensions (page 9) for maximum crankcase drain fitting depth.

## **6. HYDRAULIC FLUIDS**

Dependent on motor (see Model Code position 1) suitable fluids include:

- Antiwear hydraulic oils.
- Phosphate esters (HFD fluids)
- Water glycols (HFC fluids)
- 60/40% water-in-oil emulsions (HFB fluids)■
- 5/95% oil-in-water emulsions (HFA fluids)■
- Reduced pressure and speed limits, see page 4.

Viscosity limits when using any fluid except oil-in-water (5/95) emulsions are:

Max. off load	2000 cSt (9270 SUS)
Max. on load	150 cSt (695 SUS)
Optimum	50 cSt (232 SUS)
Minimum	25 cSt (119 SUS)

## PETROLEUM OIL RECOMMENDATIONS

The fluid should be a good hydraulic grade, non-detergent petroleum oil. It should contain anti-oxidant, anti-foam and demulsifying additives. It must contain antiwear or EP additives. Automatic transmission fluids and motor oils are not recommended.

## 7. TEMPERATURE LIMITS

Ambient min. .....-30°C (-22°F) Ambient max. ....+70°C (158°F)Max. operating temperature range

	Petroleum oil	Water- containing
Min.	-20°C (-4°F)	+10°C (50°F)
Max.*	+80°C (175°F)	+54°C (130°F)

To obtain optimum service life from both fluid and hydraulic system components, 65°C (150°F) normally is the maximum temperature except for water-containing fluids.

#### 8. FILTRATION

Full flow filtration (open circuit), or full boost flow filtration (closed circuit) to ensure system cleanliness to ISO 4406/1986 code 18/14 or cleaner.

#### 9. NOISE LEVELS

The airborne noise level is less than 66.7 dB(A) DIN (70 dB(A) NFPA) throughout the "continuous" operating envelope.

Where noise is a critical factor, installation resonances can be reduced by isolating the motor by elastomeric means from the structure and the return line installation. Potential return line resonances originating from liquid borne noise can be further attenuated by providing a return line back pressure of 2 to 5 bar (30 to 70 psi).

# 10. POLAR MOMENT OF INERTIA

Typical data

Displacement code	kg m²	lb in <sup>2</sup>	
30	0,012	40	
15	0,0094	32	

#### **11. MASS**

Approx. all models: 100 kg (220 lb)

#### **12. INSTALLATION DATA**

#### **GENERAL**

Spigot

The motor should be located by the mounting spigot on a flat, robust surface using correctly sized bolts. The diametral clearance between the motor spigot and the mounting must not exceed 0,15 mm (0.006"). If the application incurs shock loading, frequent reversing or high speed running, then high tensile bolts should be used, including one fitted bolt.

#### Bolt torque

The recommended torque wrench setting for the M18 bolts is: 312±7Nm (230±5 lbf ft)

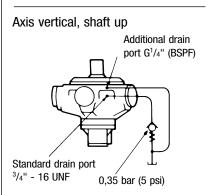
#### Shaft coupling

Where the motor is solidly coupled to a shaft having independent bearings the shafts must be aligned to within 0,13 mm (0.005") TIR.

#### CRANKCASE DRAIN

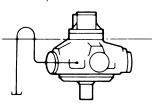
Motor axis horizontal

The crankcase drain must be taken from a position above the horizontal centre line of the motor.



An additional G¹/4" (BSPF) drain port in the front mounting flange is provided when the "V" (shaft vertically upwards) designator is given after the shaft type letter in position of the model code. This additional drain should be connected into the main motor casing drain line downstream of a 0,35 bar (5 psi) check valve to ensure lubrication of the upper bearing. See above diagram.

Axis vertical, shaft down

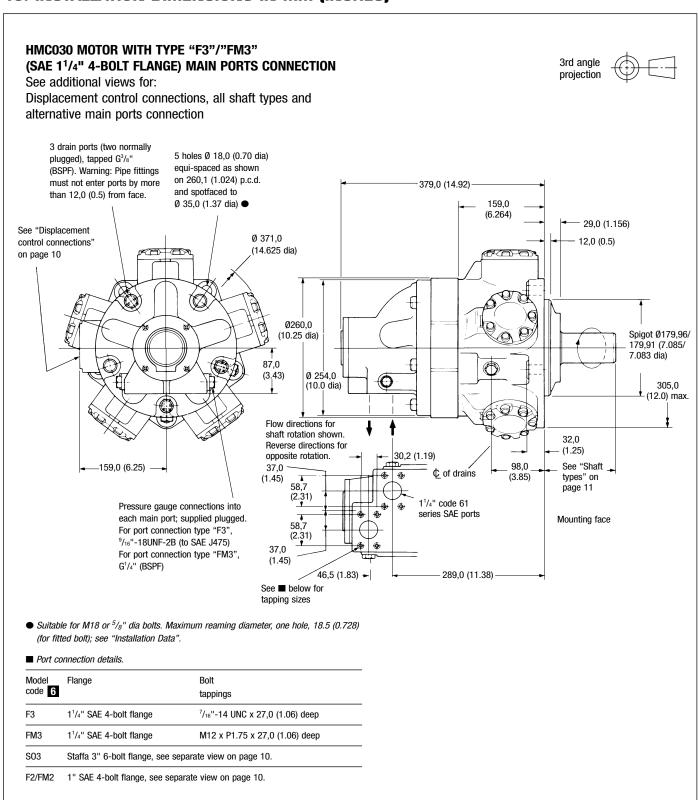


Use any drain position. The drain line should be run above the level of the uppermost bearing; if there is risk of syphoning then a syphon breaker should be fitted.

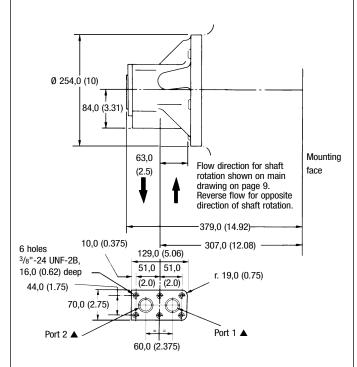
#### **START-UP**

Fill the crankcase with system fluid. Where practical, a short period (30 minutes) of "running in" should be carried out with the motor set to its high displacement (pressure to port Y, or to port B of the size 03 pilot valve).

## 13. INSTALLATION DIMENSIONS IN MM (INCHES)

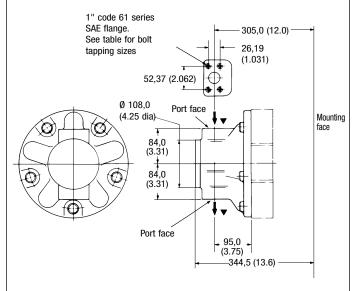


# 3" VALVE HOUSING WITH 6-BOLT FLANGE, "SO3" IN MODEL CODE POSITION 6



▲ Ø28 (1.125 dia), with recess for 31,0 (1.22) i/d x Ø 4 (0.157 dia) section 0-ring

## 1" SAE 4-BOLT FLANGE, "F2"/"FM2" IN MODEL CODE POSITION 6



#### Flange bolt tappings

Model Bolt tappings
code 6

F2 3/s"-16 UNC-2B x 22,0 (0.875) deep
FM2 M10 x P1,5 x 22,0 (0.875) deep

▼ Flow direction for shaft rotation shown on main drawing on page 9. Reverse flow for opposite direction of shaft rotation.

# DISPLACEMENT CONTROL CONNECTIONS, MODEL CODE POSITION 7

Type X G<sup>1</sup>/<sub>4</sub>" (BSPF) tapped ports X and Y

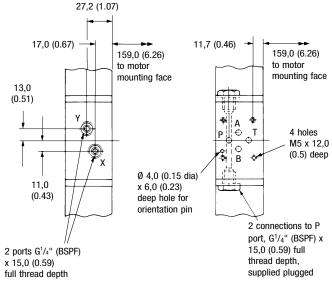
Displacement selection (via remotely located valve ◆):

High displacement: P to V: Y to T

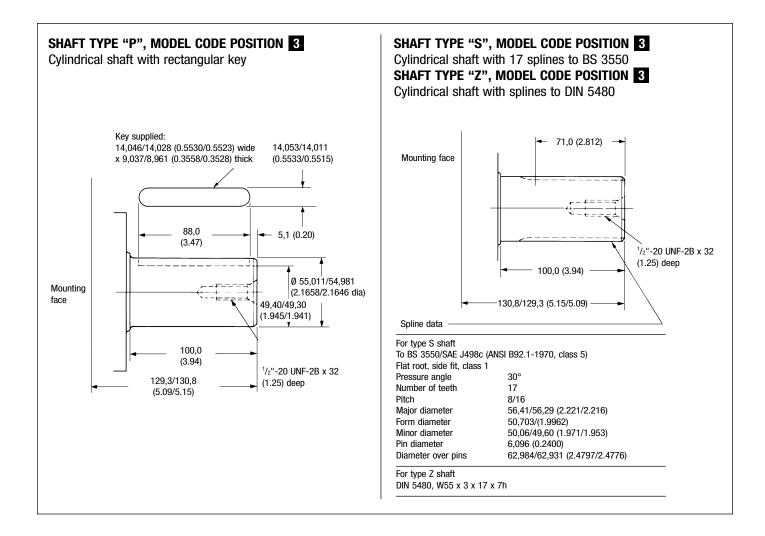
High displacement: P to Y; X to T Low displacement: P to X; Y to T

Types C, CS, C1 and C2 Mounting interface for directional control valve ◆ to: ISO 4401 size 03 ANSI/B93. 7M size D03

Displacement selection: High displacement: P to B; A to T Low displacement: P to A; B to T



 Displacement selector valve is not supplied with the motor; specify and order separately.



Presented by:



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