

Staffa Dual Displacement Hydraulic Motor



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1. GENERAL DESCRIPTION

Kawasaki "Staffa" high torque, low speed radial piston motors use hydrostatic balancing techniques to achieve high efficiency, combined with good breakout torque and smooth running capability.

The HMC series dual displacement models have two pre-set displacements which can be chosen from a wide range to suit specific application requirements. The displacements are hydraulically selected by a directional control valve which can be remote from, or mounted directly on, the motor. Displacements can be changed when the motor is running.

The range of HMC motors extends from the HMC010 of 202 cm³ (12.3 in³) to the HMC325 of 5330 cm³ (325 in³) displacement.

These motors are also available in a continuously variable version using either hydro-mechanical or electro-hydraulic control methods.

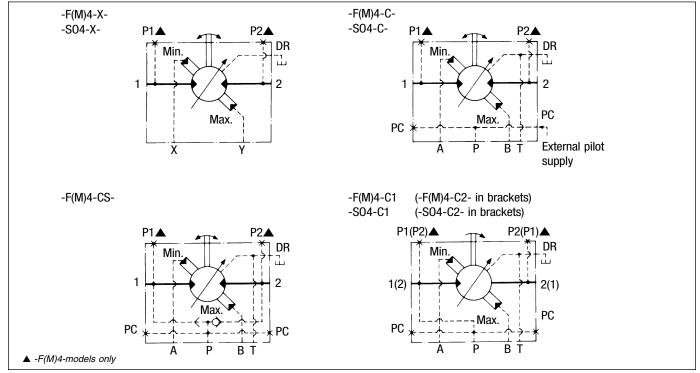
Other mounting options are available on request to match many of the competitor interfaces.

The HMC270 is one of 8 frame sizes and is capable of developing torgues up to 19 000 Nm (14 000 lbf ft) with a continuous output power of 140 kW (188 hp).

The Kawasaki "Staffa" range also includes fixed displacement motors, plus matching brakes and gearboxes to extend the torque range.

2. FUNCTIONAL SYMBOLS

All model types with variants in model code positions **6** & **7**.



3. MODEL CODE

Features shown in brackets () may be left blank according to requirements. All other features must be specified.

(F**)-HM(*)C270-**-***-***-***-(T*)-30-(PL**)

1 2 3 4 5 6 7 8 9 10

1 FLUID TYPE

- Blank = Petroleum oil
- F3 = Phosphate ester (HFD fluid)
- F11 = Water-based fluids (HFA, HFB, HFC)

2 MODEL TYPE

- Blank = Standard ("HMC")
- M = To NCB (UK) specification 463/1981 ("HMMC")

3 SHAFT TYPE

- P* = Cylindrical shaft with 2 parallel keys
- S* = Cylindrical, 20 splines to BS 3550
- Q^* = Cylindrical, internal splines to DIN 5480 (W75 x 3 x 24 x 9H)
- Z^* = Cylindrical shaft to DIN 5480 (W90 x 4 x 21 x 7h)
- * For installations where shaft is vertically upwards specify "V" after shaft type letter to ensure that additional high level drain port is provided.

4 HIGH DISPLACEMENT CODE

200 to 280 in^3 (see table 1)

5 LOW DISPLACEMENT CODE

10 to 200 in³ (see table 1)

6 MAIN PORT CONNECTIONS

- S04 = 6-bolt (UNF) flange: 4" valve (Staffa original valve housing)
- F4 = SAE $1^{1}/2^{"}$ 4-bolt (UNC, code 62) flanges: 4" valve FM4 = SAE $1^{1}/2^{"}$ 4-bolt (metric,
- code 62) flanges: 4" valve

7 DISPLACEMENT CONTROL PORTS (AND SHUTTLE VALVE)

Threaded ports/bi-directional shaft rotation:

 $X = X \text{ and } Y \text{ ports } G^{1/4"} \text{ (BSPF to ISO 228/1)}$

ISO 4401 size 03 mounting face/bidirectional shaft rotation:

C = No shuttle valve

CS = With shuttle valve

ISO 4401 size 03 mounting face/uni-directional shaft rotation (viewed on shaft end):

- C1 = Control pressure from main port 1 (shaft rotation clockwise with flow into port 1)
- C2 = Control pressure from main port 2 (shaft rotation counter-clockwise with flow into port 2)
- Not available with "SO4" type main port connections

8 TACHO/ENCODER DRIVE

- T = Staffa original tacho drive
- T1 = Suitable for Hohner 3000 series encoders. (Encoder to be ordered separately) Omit if not required.

-

9 DESIGN NUMBER, 30 SERIES

Subject to change. Installation and performance details remain unaltered for design numbers 30 to 39 inclusive.

10 SPECIAL FEATURES

PL^{**} = non-catalogued features, e.g.:

High pressure shaft seals Alternative port connections

Stainless steel shaft sleeves Alternative encoder and tacho drives Motor valve housing orientation Shaft variants

Special paint

** Number assigned as required to specific customer build.

4. PERFORMANCE DATA

Performance data is valid for Staffa HMC270 motors fully run in and operating with petroleum oil. Leakage values are at fluid viscosity of 50 cSt (232 SUS).

MOTOR SELECTION

Use table 1 to select appropriate displacements for each application. Refer to table 2 for pressure and speed limits when using fire-resistant fluids.

TABLE 1

Displacement code*

(Model code position	าร														
4 & 5)		280	250	220	200	180	160	140	120	100	80	60	40	20	10
Displacement	cm ³	4588	4097	3605	3277	2950	2622	2294	1966	1640	1310	980	655	328	160
volume/r	in ³	280	250	220	200	180	160	140	120	100	80	60	40	20	10
Average actual	Nm/bar	69,4	61,9	53,9	49,0	43,6	38,3	33,2	27,9	22,4	17,1	12,2	7,9	2,4	0
running torque	Ibf ft/psi	3.53	3.15	2.74	2.49	2.22	1.95	1.69	1.42	1.14	0.87	0.62	0.4	0.12	0
Max. continuous speed	r/min	120	135	150	165	185	200	240	270	325	350	350	350	350	1000
Max. continuous	kW	140	130	119	111	103	95	86	77	66	54	42	28	10	0
output	hp	188	174	160	149	138	127	115	103	88	72	56	38	13	0
Max. intermittent output	kW	180	175	167	160	152	143	132	120	107	95	80	55	20	0
	hp	241	235	224	214	204	192	177	161	144	127	107	74	27	0
Max. continuous pressure	bar	250	250	250	250	250	250	250	250	250	250	250	250	250	17 ♦
	psi	3626	3626	3626	3626	3626	3626	3626	3626	3626	3626	3626	3626	3626	250 ♦
Max. intermittent pressure	bar	275	275	275	275	275	275	275	275	275	275	275	275	275	17 ♦
	psi	4000	4000	4000	4000	4000	4000	4000	4000	4000	4000	4000	4000	4000	250 ♦

* Intermediate displacements in cinveements of 10 in³ are available to order.

• See "Small displacements" page 7 for information about higher pressure applications.

TABLE 2

Fluid type	Pressure, ba Continuous	r (psi) Intermittent	Max. speed r/min r/min
HFA, 5/95% oil-in-water emulsion	103 (1500)	138 (2000)	50% of limits for petroleum oil
HFB, 60/40% water-in-oil emulsion	138 (2000)	172 (2500)	As for petroleum oil
HFC, water glycol	103 (1500)	138 (2000)	50% of limits for petroleum oil
HFD, phosphate ester	250 (3626)	275 (4000)	As for petroleum oil

RATING DEFINITIONS

• CONTINUOUS RATING

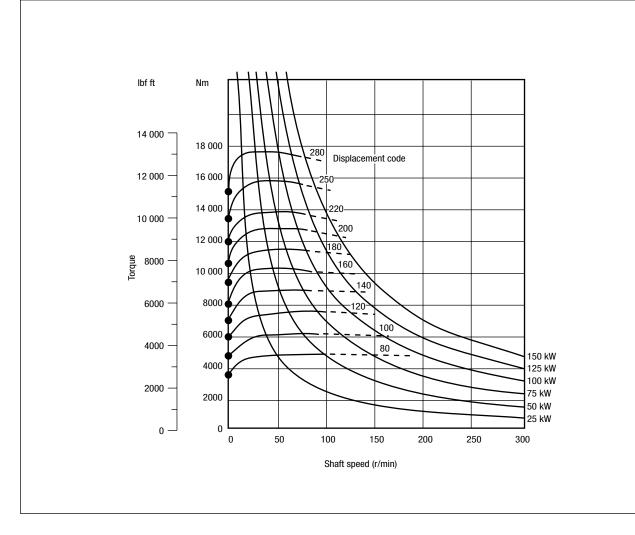
For continuous duty the motor must be operating within each of the maximum values for speed, pressure and power as specified for each displacement code.

• INTERMITTENT RATING

Operation within the intermittent power rating (up to the maximum continuous speed) is permitted on a 15% duty basis, for periods up to 5 minutes maximum.

- INTERMITTENT MAX. PRESSURE Up to 275 bar (4000 psi) is allowable on the following basis:
- (a) Up to 50 r/min: 15% duty for periods up to 5 minutes maximum.
- (b) Over 50 r/min: 2% duty for periods up to 30 seconds maximum.

OUTPUT TORQUES

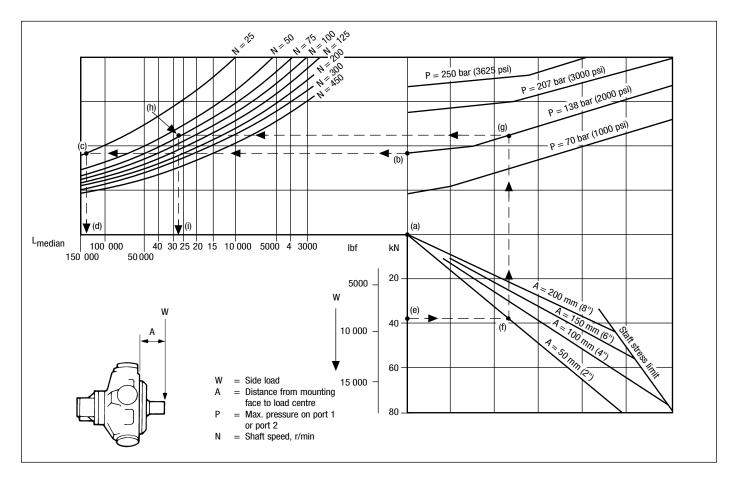


The torque curves indicate, for each displacement, the maximum output torque of the motor with an inlet pressure of 250 bar (3626 psi) and zero output pressure. High return line pressure will reduce the torque for any given pressure differential.

The solid line portion of each curve indicates the levels of maximum torque and speed that are permitted on a "continuous" basis.

The dotted portion of each curve indicates the levels of torque and speed at which the motor can operate at an "intermittent" rating.

The starting torques shown on the graph are average and will vary with crankshaft angle.



For more precise life prediction, or

where axial thrusts are incurred, a

computer analysis can be provided by

Kawasaki on receipt of machine duty

The nomograph allows the median ▲ bearing life to be determined for conditions of:

- 1. No side load and no axial thrust
- 2. Side load and no axial thrust
- ▲ To determine L10 life predictions per ISO 281-1-1977 multiply the median figure by 0.2.

	HMC270
Example 1 (follow chain dotted line):	
Side load (W)	a) 0
System pressure (P)	b) 138 bar (2000 psi)
Speed (N)	c) 25 r/min
Median bearing life	d) 130 000 hrs
L10 bearing rating = median $x 0.2$	26 000 hrs
Example 2 (follow chain dotted line):	
Side load (W)	e) 38 kN (8500 lbf)
Load offset (A) from motor mounting face	f) 50 mm (2.0 in)
System pressure (P)	g) 138 bar (2000 psi)
Speed (N)	h) 75 r/min
Median bearing life	i) 26 500 hrs
L10 bearing rating = median $x 0.2$	5300 hrs

cycle.

 SHAFT STRESS LIMIT
The shaft stress limit in the nomograph is based on the fatigue rating of the shaft. Infrequent loading above these limits may be permitted; consult Kawasaki.

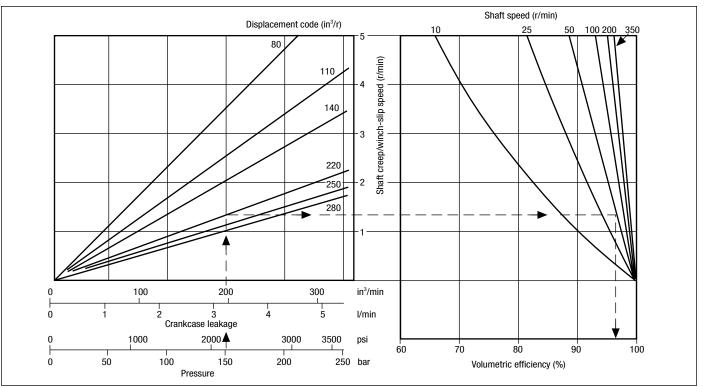
VOLUMETRIC EFFICIENCY

The nomograph on page 7 enables the average volumetric efficiency, crankcase (drain) leakage and "winch slip"/shaft creep speed to be estimated.

Example (follow chain dotted line): Given:

1. Pressure	150 bar (2175 psi)
2. Displacement cod	e220 (in ³ /r)
3. Speed	50 r/min
To obtain:	
4. Volumetric efficier	1cy96.8%
5. Crankcase leakage	e3,2 l/min
	(195 in ³ /min)
6. Shaft creep speed	1.3 r/min

The shaft creep occurs when the load attempts to rotate the motor against closed ports as may occur, for example, in winch applications.



5. CIRCUIT AND APPLICATION NOTES

DISPLACEMENT SELECTION

To select either displacement, a pressure at least equal to 2/3 of the motor inlet/outlet pressure (whichever is higher) is required. In most applications the motor inlet pressure will be used.

For inlet/outlet pressures below 3,5 bar (50 psi) a minimum control pressure of 3,5 bar (50 psi) is required. In the event of loss of control pressure the motor will shift to its highest displacement.

For rapid reversing applications it is recommended to externally source the control oil supply direct from the system pump (use displacement control type "X" or "C" - *not* "CS", "C1" or "C2" - in model code position **1**).

STARTING TORQUES

The starting torques shown on the graph on page 5 are average and will vary with system parameters. For motors with low displacement below 80 in³ and starting under load it is recommended to select high displacement for start-up.

LOW SPEED OPERATION

(High displacement mode) Minimum operating speeds are determined by load conditions (load inertia, drive elasticity, etc.) For operation at speeds below 3 r/min consult Kawasaki.

SMALL DISPLACEMENTS

(10 in³ and below)

The pressures given in the table on page 4 for displacement code "10" (and below) are based on 1000 r/min output shaft speed. These pressures can be increased for shaft speeds less than 1000 r/min; consult Kawasaki for details.

In addition to 10 in³, a zero swept volume displacement (for free wheeling requirements) is available on request, subject to Kawasaki approving the application.

HIGH BACK PRESSURE

When both inlet and outlet ports are pressurized continuously, the lower pressure in one port must not exceed 70 bar (1000 psi). Consult Kawasaki on applications beyond this limit. Note that high back pressures reduce the effective torque output of the motor.

BOOST PRESSURE

When operating as a motor the outlet pressure should equal or exceed the crankcase pressure. If pumping occurs

(i.e. overrunning loads) then a positive pressure, "P", is required at the motor ports. Calculate "P" (bar/psi) from the appropriate formula:

P (bar) = 1 +
$$\frac{N^2 \times V^2}{4 \times 10^{10}}$$
 + C

Where:

- C = crankcase pressure, bar
- N = speed, r/min
- V = displacement, cm³/r

P (psi) =
$$14.5 + \frac{N^2 \times V^2}{1 \times 10^7} + C$$

Where:

- C = crankcase pressure, psi
- N =speed, r/min
- $V = displacement, in^3/r$

The flow rate of oil needed for the make-up system can be estimated from the crankcase leakage figure (see Volumetric Efficiency graph above) plus an allowance for changing displacement; e.g. to change high to low in 1 sec requires 24 l/min (6.5 USgpm).

Allowance should be made for other system losses and also for "fair wear and tear" during the life of the motor, pump and other system components.

COOLING FLOW

Operation within the continuous ratings does not require any additional cooling.

For operating conditions above "continuous", up to the "intermittent" ratings, additional cooling oil may be required. This can be introduced through the spare crankcase drain hole, or in special cases through the valve spool end cap. Consult Kawasaki about such applications.

MOTOR CASING PRESSURE

With the standard shaft seal fitted, the motor casing pressure should not exceed 3,5 bar (50 psi). On installations with long drain lines a relief valve is recommended to prevent over-pressurizing the seal.

Notes:

- 1. The casing pressure at all times must not exceed either the motor inlet or outlet pressure.
- 2. High pressure shaft seals are available to special order for casing pressures of: Continuous: 10 bar (150 psi) Intermittent: 15 bar (225 psi)
- 3. Check installation dimensions (page 9) for maximum crankcase drain fitting depth.

6. HYDRAULIC FLUIDS

Dependent on motor (see Model Code position 1) suitable fluids include:

- Antiwear hydraulic oils
- Phosphate esters (HFD fluids)
- Water glycols (HFC fluids)
- 60/40% water-in-oil emulsions (HFB fluids)■
- 5/95% oil-in-water emulsions (HFA fluids)■
- Reduced pressure and speed limits, see page 4.

Viscosity limits when using any fluid except oil-in-water (5/95) emulsions are: Max off load 2000 cSt (9270 SUS)

Mux. on louu	
Max. on load	
Optimum	
Minimum	

PETROLEUM OIL RECOMMENDATIONS

The fluid should be a good hydraulic grade, non-detergent petroleum oil. It should contain anti-oxidant, anti-foam and demulsifying additives. It must contain antiwear or EP additives. Automatic transmission fluids and motor oils are not recommended.

7. TEMPERATURE LIMITS

Ambient min.	30°C (-22°F)
Ambient max.	+70°C (158°F)

Max. operating temperature range

	Petroleum oil	Water- containing
Min.	-20°C (-4°F)	+10°C (50°F)
Max.*	+80°C (175°F)	+54°C (130°F)

* To obtain optimum service life from both fluid and hydraulic system components 65°C (150°F) normally is the maximum temperature except for water-containing fluids.

8. FILTRATION

Full flow filtration (open circuit), or full boost flow filtration (closed circuit) to ensure system cleanliness of ISO 4406/1986 code 18/14 or cleaner.

9. NOISE LEVELS

The airborne noise level is less than 66.7 dB(A) DIN (70 dB(A) NFPA) throughout the "continuous" operating envelope.

Where noise is a critical factor, installation resonances can be reduced by isolating the motor by elastomeric means from the structure and the return line installation. Potential return line resonances originating from liquid borne noise can be further attenuated by providing a return line back pressure of 2 to 5 bar (30 to 70 psi).

10. POLAR MOMENT OF INERTIA

Typical data

Displacement code	kg m²	lb in ²	
280	0,83	2850	
100	0,61	2100	

11. MASS

Approx. all models: 450 kg (1000 lb)

12. INSTALLATION DATA

GENERAL

Spigot

The motor should be located by the mounting spigot on a flat, robust surface using correctly sized bolts. The diametral clearance between the motor spigot and the mounting must not exceed 0,15 mm (0.006"). If the application incurs shock loading, frequent reversing or high speed running, then high tensile bolts should be used, including one fitted bolt.

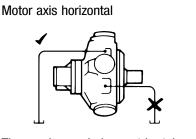
Bolt torque

The recommended torque wrench settings for the mounting bolts are: M20407 \pm 14 Nm (300 \pm 10 lbf ft) ³/4"393 \pm 14 Nm (290 \pm 10 lbf ft)

Shaft coupling

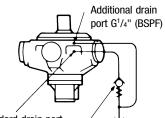
Where the motor is solidly coupled to a shaft having independent bearings the shafts must be aligned to within 0,13 mm (0.005") TIR.

CRANKCASE DRAIN



The crankcase drain must be taken from a position above the horizontal centre line of the motor.

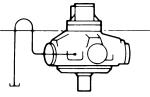
Axis vertical, shaft up



Standard drain port ³/₄" - 16 UNF 0,35 bar (5 psi)

An additional G¹/4" (BSPF) drain port in the front cover is provided when the "V" (shaft vertically upwards) designator is given after the shaft type letter in position **3** of the model code. This additional drain should be connected into the main motor casing drain line downstream of a 0,35 bar (5 psi) check valve to ensure lubrication of the upper bearing, see above diagram.

Axis vertical, shaft down

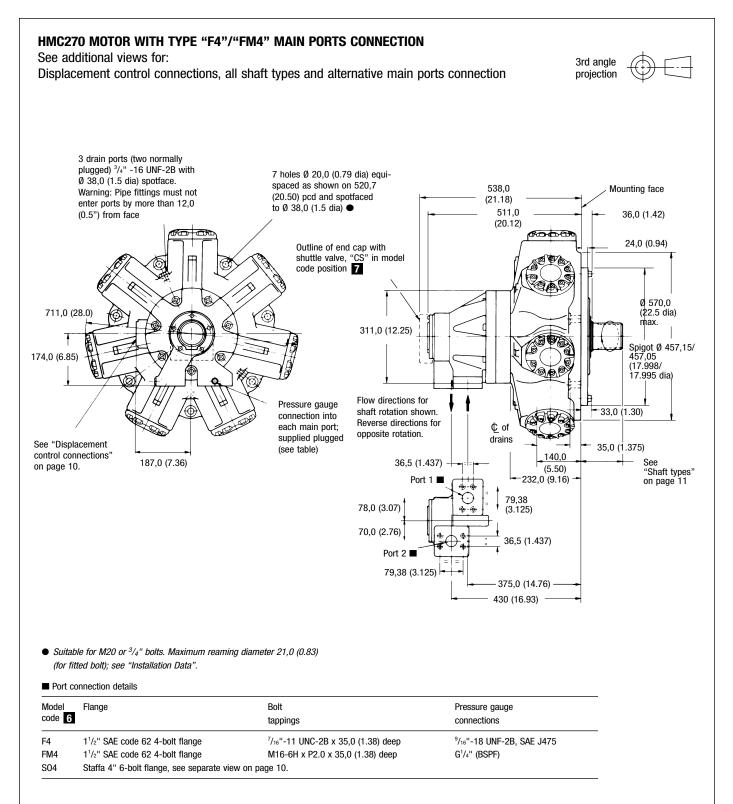


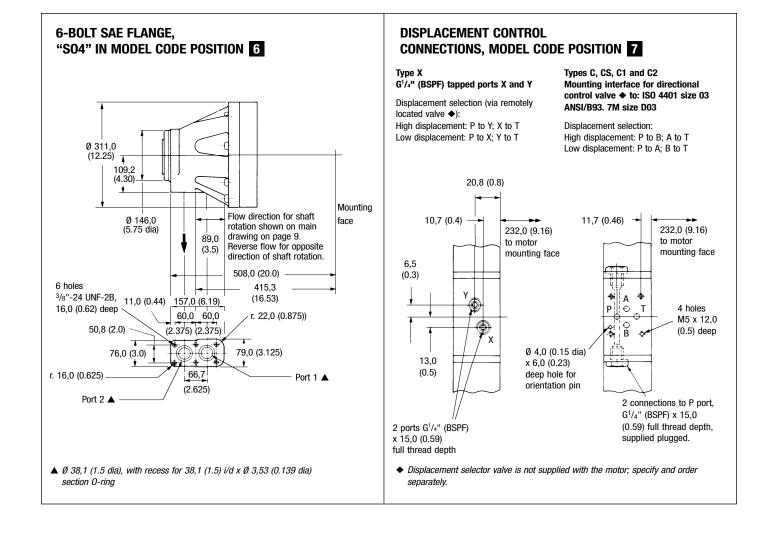
Use any drain position. The drain line should be run above the level of the uppermost bearing; if there is risk of syphoLning then a syphon breaker should be fitted.

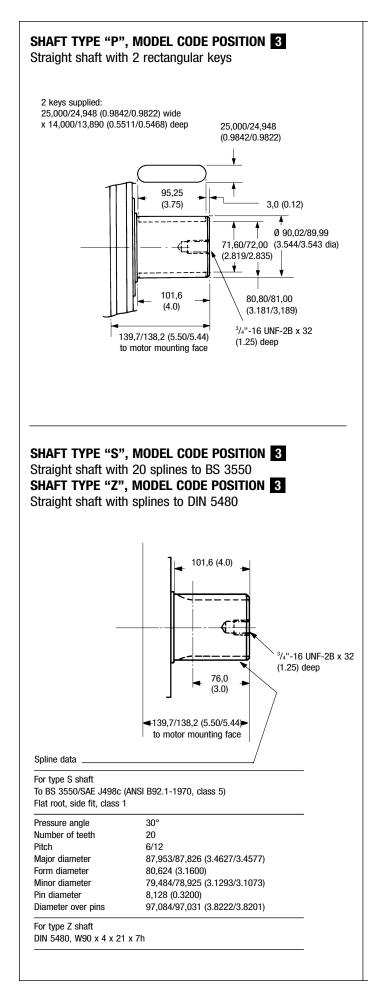
START-UP

Fill the crankcase with system fluid. Where practical, a short period (30 minutes) of "running in" should be carried out with the motor set to its high displacement (pressure to port Y, or to port B of the size 03 pilot valve).

13. INSTALLATION DIMENSIONS IN MM (INCHES)



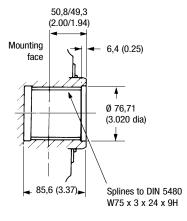




SHAFT TYPE "Q", MODEL CODE POSITION 3

Cylindrical, internal splines to DIN 5480

Note: The "Q" shaft will transmit the maximum torques given on page 5. However, customers should ensure that their own mating shaft will transmit the torque required in their application.



Presented by:



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